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UNITED STATES PATENT APPLICATION

Title:

COAXIAL STEERING AND SUSPENSION FOR MOTORCYCLE

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COAXIAL STEERING AND SUSPENSION FOR MOTORCYCLE

Background of the Invention

Technical Field of the Invention

This invention relates generally to suspension components for two-wheeled vehicles, and more specifically to front steering and front spring/shock components.

Background Art

The vast majority of motorcycles (and full- and front-suspension bicycles) are equipped with front suspensions in which a pair of telescopic forks are coupled to the steering head of the motorcycle's frame by an upper triple clamp and a lower triple clamp. The triple clamps provide enough lateral offset that the forks clear the sides of the front tire. The triple clamps are usually constructed to provide some measure of longitudinal offset, as well, to increase trail and thereby stabilize the motorcycle. Trail is a measurement, on the ground, from a point projected through steering axis to the center of the tire's contact patch directly below the axle, and determines the self-centering stability of the steering. The forks are either of the "right-side-up" or sliding-female configuration, or the "upside-down" or sliding-male configuration. In either case, a cylindrical tube or piston slides axially within a cylindrical cylinder.

In nearly all cases, both the suspension springs and the damping or shock absorbing components are disposed within one or both of the sliding tube assemblies. Unfortunately, because the substantial mass of the springs, dampers, oil, and other related suspension components is located a significant distance – generally in the neighborhood of 2.5 inches – from the axis of the steering head, the front end has an undesirably large moment of rotational inertia. In other words, the front end has a heavy "swing weight" which reduces both the "feel" and the responsiveness of the front end.

Alternative front end configurations have occasionally been seen, but all suffer from this same malady, and their inventors have been attempting to solve other problems, such as front end "dive" under hard braking, rather than reducing the moment of rotational inertia. Examples include the well-known BMW Telelever, the Britten front end, the Hossack front end, the RADD-Yamaha front end, and various hub-center systems such as that found on the Bimota Tesi.

None of these previous geometries places the spring or damper components coaxial to the steering head, and all suffer from having stylistic, aesthetic appearances which are dramatically different than the almost universally preferred conventional dual fork system. Furthermore, all are significantly more complex than the conventional dual fork system. The downside of these previous systems, such as increased mass, outweigh any benefit they may have offered on other fronts.

Fork flex, especially under braking, is a significant contributor to the stiction which is known to dramatically reduce the effectiveness and perceived quality of a motorcycle's front suspension.

What is needed, then, is a system which has the aesthetic appeal and simplicity of the dual fork geometry, with a significantly reduced moment of rotational inertia. What is further needed is a system which offers reduced stiction.

Brief Description of the Drawings

The invention will be understood more fully from the detailed description given below and from the accompanying drawings of embodiments of the invention which, however, should not be taken to limit the invention to the specific embodiments described, but are for explanation and understanding only.

FIGS. 1-4 show various views of one embodiment of the front end and steering/suspension system of the present invention.

FIG. 5 shows an exploded view of the steering/suspension system.

FIG. 6 shows a perspective, cutaway view of a portion of the steering/suspension system.

FIG. 7 shows a motorcycle having the coaxial steering and suspension front end of this invention.

Detailed Description

FIGS. 1-4 illustrates one embodiment of a motorcycle front end 10 and specifically the steering/suspension system 50, viewed generally from the front in FIGS. 1 and 2, the left side in FIG. 3, and the rear in FIG. 4. The front end includes a tire 12 mounted on a wheel 14 and equipped with brake rotors 16 and brake calipers 18. A fork bottom 20 includes a fork bottom body coupled to the axle 22 and to a pair of male lower fork tubes 24 which slide in a pair of

1 female upper fork tubes 26. In other embodiments, conventional fork tubes are used, without
2 fork bottoms. A lower triple clamp 28 and an upper triple clamp 30 are coupled to the upper fork
3 tubes and couple them to a steering stem assembly (not visible) which rotates within a steering
4 tube 32 which is part of, or coupled to, the frame (not shown) of the motorcycle.

5 The front end 10 thus pivots or rotates about a steering axis which is coaxial with the
6 steering tube 32. This invention differs from the prior art in that at least one of the spring
7 (suspension) and/or shock (damping) components is coaxially disposed within the steering tube.
8 In one embodiment, a monoshock 34 provides both spring support and damping for the front
9 end, while in other embodiments, a more conventional cartridge system (not shown) could be
10 employed within the fork tubes. The bottom end of the monoshock is coupled to a fork buttress
11 36. The fork buttress may be coupled to the lower fork tubes or to the fork bottoms. In one
12 embodiment, the fork buttress comprises two halves, each of which is integrally formed with a
13 respective fork bottom, as shown.

14 FIG. 5 illustrates the steering/suspension system 50 of the motorcycle front end 10,
15 viewed generally from the front and shown in an exploded view. For ease of illustration, only a
16 single fork will be described. The upper fork tube 26 threads into the upper triple clamp 30. A
17 fork cap 52 seals the open end of the fork tube to prevent gross contamination of the sliding
18 components, but is not necessarily an airtight seal. A stationary fork bushing 54 and a seal 56 fit
19 within the lower end 48 of the upper fork tube, and are held in place by a snap wire 58. A sliding
20 fork bushing 60 mates with the upper end 62 of the lower fork tube. The stationary and sliding
21 fork bushings provide a low-stiction but tight-tolerance sliding fit of the lower fork tube within
22 the upper fork tube. In practice, the components may need to be assembled in a slightly different
23 manner than suggested by this exploded view, as the bushings are not generally able to slide past
24 each other, and their interference is part of what keeps the telescopic forks from telescoping
25 completely to disassembly.

26 A shock tube 66 rotates within the steering tube 32 on an upper bearing 68 and a lower
27 bearing 70. A jam nut 72 and washer 74 secure the upper bearing onto the shock tube. A top bolt
28 76 threads into the shock tube and secures it to the upper triple clamp.

29 The lower end 78 of the lower fork tube threads or otherwise couples to a hole 80 in the
30 upper end 42 of the fork bottom 20. The upper end of the monoshock 34 fits up into and engages

1 the shock tube, while the lower end of the monoshock engages the fork buttress 36 at the upper
2 end 42 of the fork bottom.

3 The lateral stiffness of the fork bottom is controlled by a tension cable 82. The lower end
4 84 of the tension cable engages the fork bottom, while the upper end 86 of the tension cable is
5 engaged and tensioned by a tension adjuster 88. The tension adjuster and a washer 90 engage a
6 tension adjuster block 92 which fits into a hole 94 in the upper end of the fork bottom. A detent
7 ball 96 retains the tension adjuster within the tension adjuster block and, in some embodiments,
8 provides "clicker" adjustment feedback as is commonly present in other motorcycle suspension
9 adjustments such as compression and rebound damping. For aesthetics and aerodynamics, a fork
10 bottom inner cover 98 may be coupled to the inward portion of the fork bottom, covering the
11 tension cable and other components.

12 FIG. 6 illustrates further details of one embodiment of the steering/suspension system 50,
13 with a cutaway for visibility into the coaxial alignment of the suspension components within the
14 steering tube 32. The suspension components are illustrated somewhat generically and in a much
15 simplified configuration omitting many details which are not essential to understanding this
16 invention but which are well within the abilities of those of ordinary skill in the art. The
17 suspension components may include one or more load-bearing components such as a coil spring
18 110, and one or more damping components 112. As such, the suspension components may be
19 quite similar to a conventional rear shock such as is conventionally used in modern sportbikes,
20 with the addition of a suitable mounting mechanism 114 adapted for coupling or mating with the
21 fork brace (not shown) or other lower mounting component.

22 The suspension components are disposed coaxially with the steering tube 32, or, more
23 precisely, coaxially with the steering axis. The shock tube 66 is disposed coaxially within the
24 steering tube, and rides on an upper bearing 68 and a lower bearing 70. The jam nut 72 is
25 threaded onto the shock tube. The top bolt 76 threads into the shock tube and coaxially locates
26 the upper triple clamp 30 with respect to the steering axis. The top bolt is provided with, in one
27 embodiment, an internal hex socket 116 by which the top nut is tightened.

28 In one embodiment, the top nut is further provided with a passage 118 and the shock tube
29 is provided with a passage, through which a tool (not shown) can be inserted to adjust various
30 settings of the suspension components, such as compression damping, rebound damping,

1 preload, ride height, and so forth. Again, for ease of illustration, these various adjustment
2 mechanisms are not shown on the monoshock.

3 One noteworthy feature of this system is that the ride height of the front end can be
4 adjusted by screwing threaded rod 120 up and down in the shock tube, and this is completely
5 independent of the coupling of the forks to the triple clamps. This represents a marked
6 improvement over the conventional fork systems, in which the rider must loosen the upper and
7 lower triple clamps, slide or pound the upper fork tubes up and down in the triple clamps until a
8 desired amount of protrusion is achieved, then retighten the triple clamps, while hoping that the
9 fork tubes have not shifted and that the two fork tubes are set at exactly the same height. The
10 coaxial monoshock adjustment of this invention enables the rider to adjust the ride height
11 without fiddling with the triple clamps or fork tubes, and it guarantees a single, consistent setting
12 without the possibility of maladjustment between the two forks. The same monoshock principle
13 applies to other adjustments, as well, such as compression damping, rebound damping, and so
14 forth.

15 FIG. 7 illustrates a motorcycle 150 having a front end with the coaxial
16 steering/suspension system of this invention.

17 Conclusion

18 The skilled reader will readily appreciate that having the suspension components
19 mounted coaxially with the steering head provides several significant advantages. For example:
20 the moment of rotational inertia of the front end is reduced, versus that of a conventional front
21 end in which the suspension components are located out in the fork tubes; only a single set of
22 suspension components is required, and yet the suspension has the same affect at each side of the
23 front axle, whereas putting a single set of components in e.g. only the left fork of a conventional
24 front end would produce disastrous results; preload, rebound damping, compression damping,
25 and ride height adjustments can be made with a single adjustment each, versus the two
26 adjustments each that are required in a conventional front end, and can be done without
27 loosening the forks in the triple clamps; suspension settings cannot accidentally be different on
28 the two sides of the front end, whereas this is a constant danger with a conventional front end;
29 stiction is reduced; and yet the familiar and desirable look and feel of a conventional dual fork

1 front end are retained. Furthermore, it may often be the case that the total mass of the required
2 suspension is lower when using the present invention, than when using a conventional front end.

3 While the invention has been described with reference to its use in a motorcycle, the
4 invention is not limited to motorcycles, but can be used in bicycles, automobiles, and other
5 vehicles. And while the invention has been shown as using an “upside-down” fork, it may
6 alternatively be used with a “right-side-up” fork. Some components have been illustrated as
7 being of monolithic construction, while other components have been illustrated as being separate
8 components coupled together. The skilled reader will readily appreciate that the designer may
9 elect, within the scope of this invention, to split some components into separate sub-components,
10 or to combine various components into a monolithic whole. The skilled reader will further
11 appreciate that the invention may be practiced in a “single-sided” front end, such as that found
12 on some bicycles which have only a single fork. The term “triple clamp” should not necessarily
13 be interpreted to mean that two forks are required with the steering tube. The presence of one or
14 more suspension components coaxial with the steering axis does not necessarily prohibit the
15 additional presence of one or more suspension components elsewhere, such as within the forks.

16 The sliding-tube forks may be empty, containing neither springs nor dampers, and may
17 thus be said to have substantially inert suspension characteristics. In some embodiments, the
18 suspension components could be located externally to the outer steering tube, rather than inside
19 it.

20 While it might, at first glance, be assumed to be a negative that the steering tube must, in
21 the present invention, be significantly larger than in a conventional front end, the opposite is
22 actually true. Having a very large diameter steering tube, with very large diameter bearings and
23 so forth, reduces frame torque and makes the frame stronger, especially at the points at which the
24 rest of the frame joins the steering tube.

25 When one component is said to be “adjacent” another component, it should not be
26 interpreted to mean that there is absolutely nothing between the two components, only that they
27 are in the order indicated. The various features illustrated in the figures may be combined in
28 many ways, and should not be interpreted as though limited to the specific embodiments in
29 which they were explained and shown. Those skilled in the art having the benefit of this
30 disclosure will appreciate that many other variations from the foregoing description and

1 drawings may be made within the scope of the present invention. Indeed, the invention is not
2 limited to the details described above. Rather, it is the following claims including any
3 amendments thereto that define the scope of the invention.